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## Introduction

- Composite materials are used extensively in the aerospace industry
- Provide weight reduction and strength and stiffness tailoring
- Tow-steering allows local stiffness tailoring and possibly more weight reduction
- Also introduces local mechanical coupling
- Buckling load is an important design consideration
- Rayleigh-Ritz method has been used to predicted

## Rayleigh Ritz Method

- Assume displacement functions  
 $u(x, y), v(x, y), w(x, y)$
- Minimize total potential energy ( $\Pi$ )

$$\Pi = U + V$$

- Plate's strain energy:  $U$
- Energy from external forces:  $V$

$$\frac{\partial \Pi}{\partial S_{mn}} = 0$$

- $S_{mn}$  are unknowns in  $u, v, w$
- Solve eigenproblem  
 $([L] - N_{crit}[R])\{S\} = \{0\}$
- Eigenvalue – Critical buckling load
- Eigenvector – Mode shape

## Isotropic Case

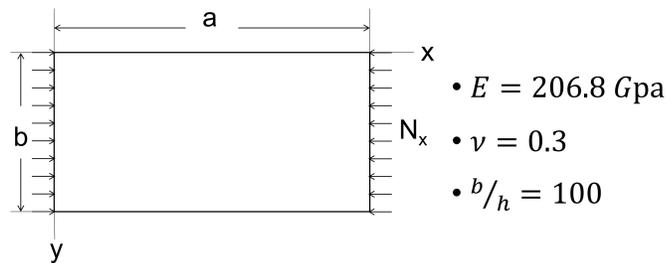
- For unsymmetric cross-ply laminates:

$$u = \sum_m \sum_n U_{mn} \cos\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right)$$

$$v = \sum_m \sum_n V_{mn} \sin\left(\frac{m\pi x}{a}\right) \cos\left(\frac{n\pi y}{b}\right)$$

$$w = \sum_m \sum_n W_{mn} \sin\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right)$$

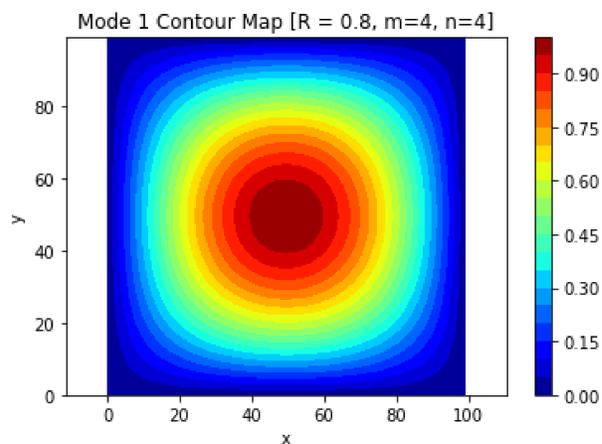
- Timoshenko and Gere<sup>1</sup> simply supported steel plate under uniform axial compression



- Results compared with nondimensional buckling coefficient  $k$

Nondimensional buckling coefficient  $k$  for various aspect ratios

$a/b$	0.4	0.6	0.8	1.0
Timoshenko & Gere	8.41	5.14	4.20	4.00
Predicted	8.41	5.14	4.20	4.00



## Mechanical Coupling

- Complicates buckling analysis and requires more accurate assumed displacement functions

- For antisymmetric angle-ply

$$u = \sum_m \sum_n U_{mn} \sin\left(\frac{m\pi x}{a}\right) \cos\left(\frac{n\pi y}{b}\right)$$

$$v = \sum_m \sum_n V_{mn} \cos\left(\frac{m\pi x}{a}\right) \sin\left(\frac{n\pi y}{b}\right)$$

$$w = w_{cross-ply}$$

- Assumed displacement function

$$u = U_1 \frac{x}{a} + u_{cross-ply} + u_{angle-ply}$$

$$v = V_1 \frac{x}{a} + V_2 \frac{y}{b} + v_{cross-ply} + v_{angle-ply}$$

$$w = w_{cross-ply} = w_{angle-ply}$$

- Compared assumed displacement functions on different laminates<sup>2</sup>
  - Hercules AS4/3501-6 graphite/epoxy
  - Isolated certain coupling terms

Laminates and associated coupling terms

Laminate	Coupling Terms
Timoshenko	None
$[0_3/90_3/0_3/90_3]$	$B_{11}, B_{12}, B_{22}$
$[0_2/45_2/0_2/45_2/0_2]$	$A_{16}, A_{26}, D_{16}, D_{26}$
$[0_2/45_2/0_2/-45_2/0_2]$	$B_{16}, B_{26}$
$[0_6/45_6]$	Full ABD

Nondimensional buckling coefficient  $k$

Laminate	Bogetti	Cross-Ply	Angle-Ply
Timoshenko	4.00	4.00	4.00
$[0_3/90_3/0_3/90_3]$	1.02	1.02	1.16
$[0_2/45_2/0_2/45_2/0_2]$	1.30	1.30	1.30
$[0_2/45_2/0_2/-45_2/0_2]$	1.13	1.28	1.13
$[0_6/45_6]$	0.78	0.92	1.06

## Future Work

- Finalize validation for mechanical coupling cases

- Rework problem to solve for  $u$  and  $v$  to compare to assumed displacement functions

- Incorporate tow-steering

$$\theta(x) = \begin{cases} 2(T_0 - T_1) \left(\frac{x}{a}\right) + T_1 & 0 \leq x \leq \frac{a}{2} \\ 2(T_1 - T_0) \left(\frac{x - \frac{a}{2}}{a}\right) + T_0 & \frac{a}{2} < x \leq a \end{cases}$$

- Fiber angle at panel center:  $T_0$
- Fiber angle at panel ends:  $T_1$
- Investigate Rayleigh-Ritz effect on capturing local mechanical coupling

## References

- Timoshenko and Gere, *Theory of Elastic Stability*. 1961<sup>1</sup>
- David Jensen, *Buckling and Postbuckling Behavior of Unbalanced and Unsymmetric Laminated Graphite/Epoxy Plates*. 1986<sup>2</sup>
- Papazoglou, *Buckling of unsymmetric laminates under linearly varying, biaxial in-plane loads, combined with shear*. 1992

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